Right-Sized HVAC Design

The Missing Ingredient in Most Homes

IECC LLC

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The “House is a System”

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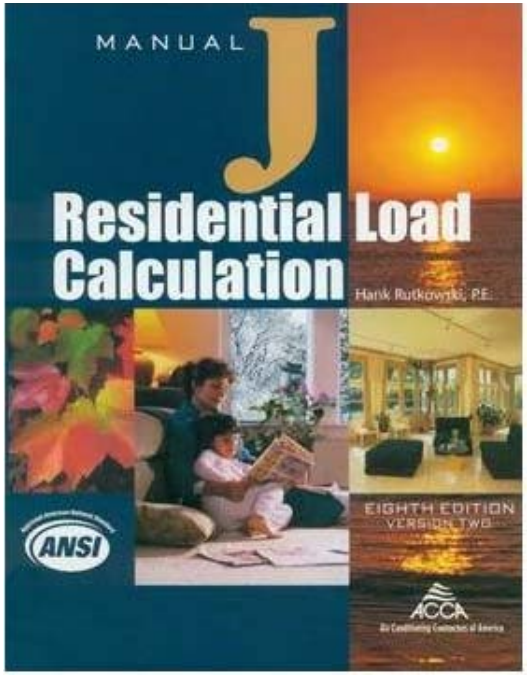
- HVAC are subsystems;
- HV & AC “condition air” for human comfort, duh!
- HVAC move and circulate air;
- HVAC affects building pressure;
- HVAC sustains life in the home (i.e., makes the house come alive)

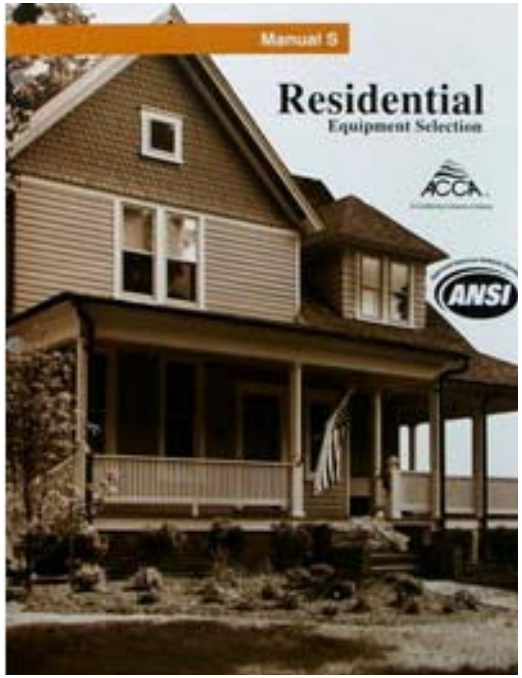
If HVAC is the backbone of a “House as a System,” then:

- The load calculation is the backbone of the HVAC system
- Required by Code (IRC, IECC)
- Necessary to:
  - Select HVAC equipment size;
  - Determine supply CFM by room
  - Select supply outlets;
  - Select return inlets;
  - Design the duct system
  - Diagnose client comfort issues

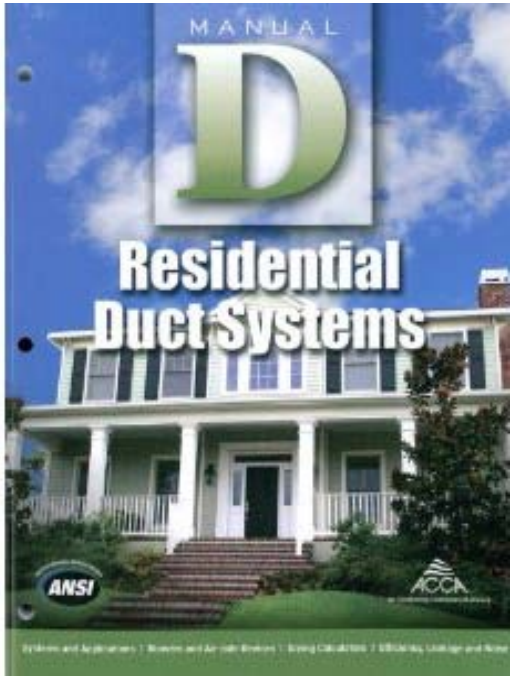
Standards of the Industry



Load Calcs

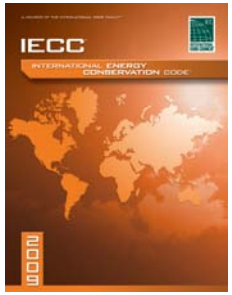


System Size



Duct Design

Why Manual J, S, D Training? What's Changed?

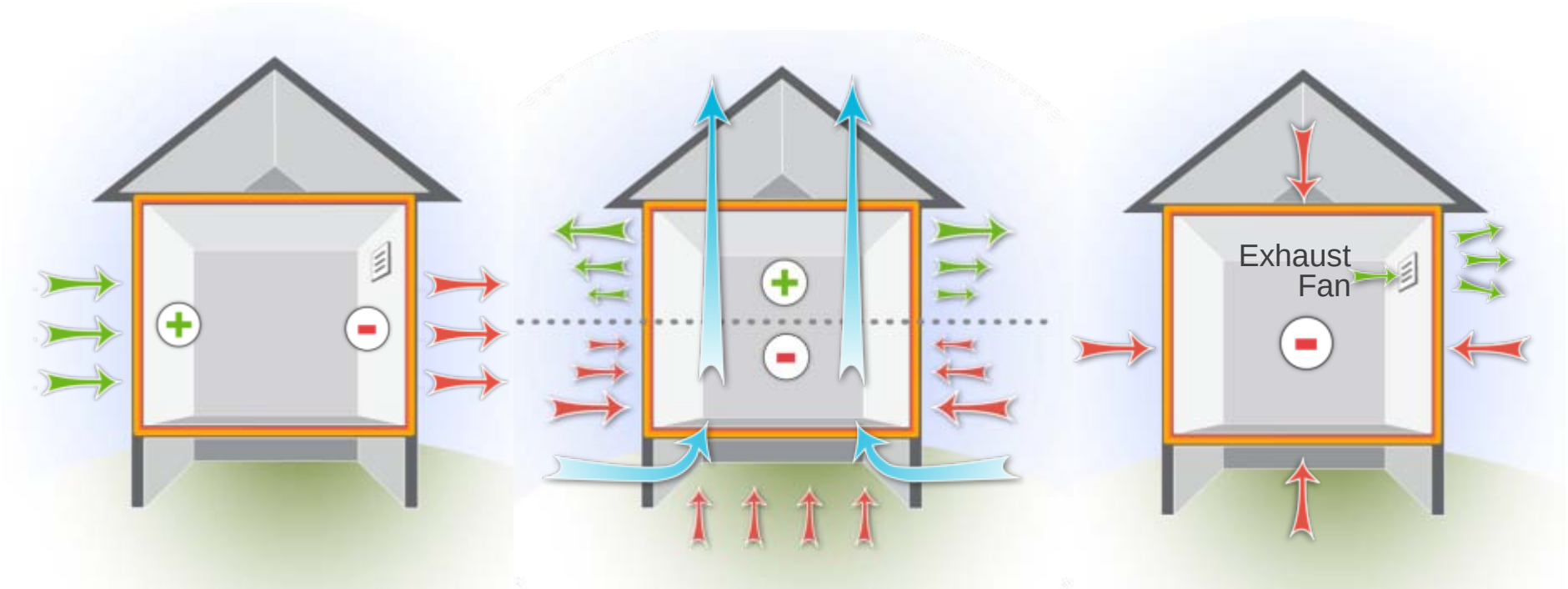


- For most in the Great Plains, 2009 IECC in force.
  - Blower door test (7ACH50) is optional to visual inspection,
  - Duct pressure testing (8 cfm/100 ft^2 of CFA), and
- For Illinois, 2012 IECC effective, January 1, 2013.
  - Mandatory Blower door test (5ACH50),
  - All *"hard-ducted"* w/ duct pressure testing (4 cfm/100 ft^2 of CFA), and
  - Mandatory whole-house ventilation (ASHRAE 62.2)
- For the average Illinois home: (2-story, 3-bdrm, 2,200 ft^2 w/bsmt)
  - A -33% reduction in envelope air-leakage (155/105 CFMn)
  - A deliberate mechanical ventilation strategy (60/240 CFMn)
  - A net reduction of 250 to 400 CFMn across the building envelope.

Why Manual J, S, D Training?

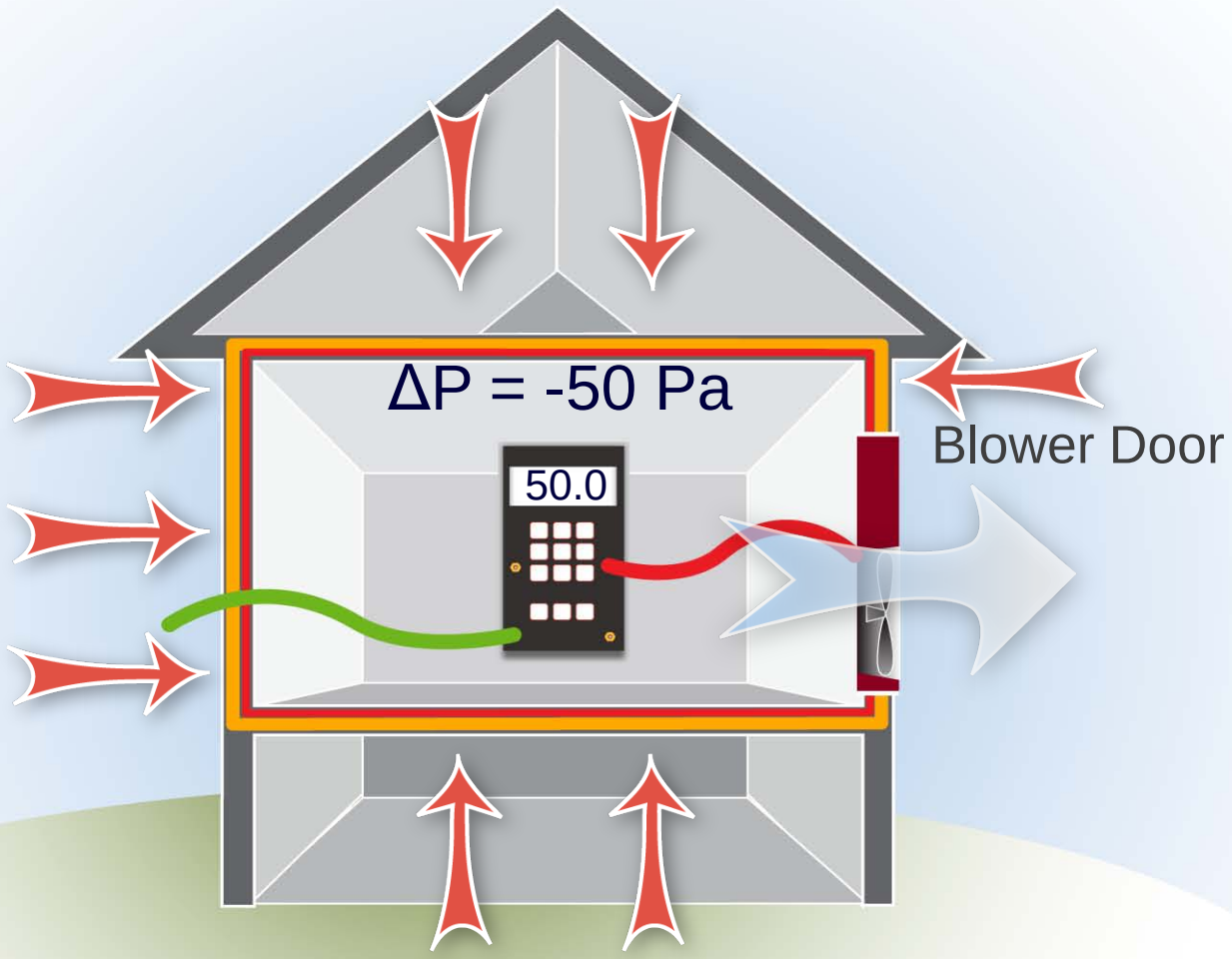
- Load estimating (MJ8), equipment sizing (MS), and duct-design (MD) concepts must be re-introduced to builders, contractors, and home owners.
- Why? They do not readily recognize that equipment size, and therefore installation and operating costs, dramatically decrease and long term comfort increases when load estimates are no longer based on rules-of-thumb and record-setting weather conditions.
- **NOTE:** *The amount of effort devoted to the design work is closely tied to the designer's ability to charge for this work.*

How Air Moves Through Buildings



Pressure ***(Wind)*** **Convection** ***(stack effect)*** **Mechanical** ***(fans)***

Blower Door Testing



R402.4.1.1 – Blower Door Testing

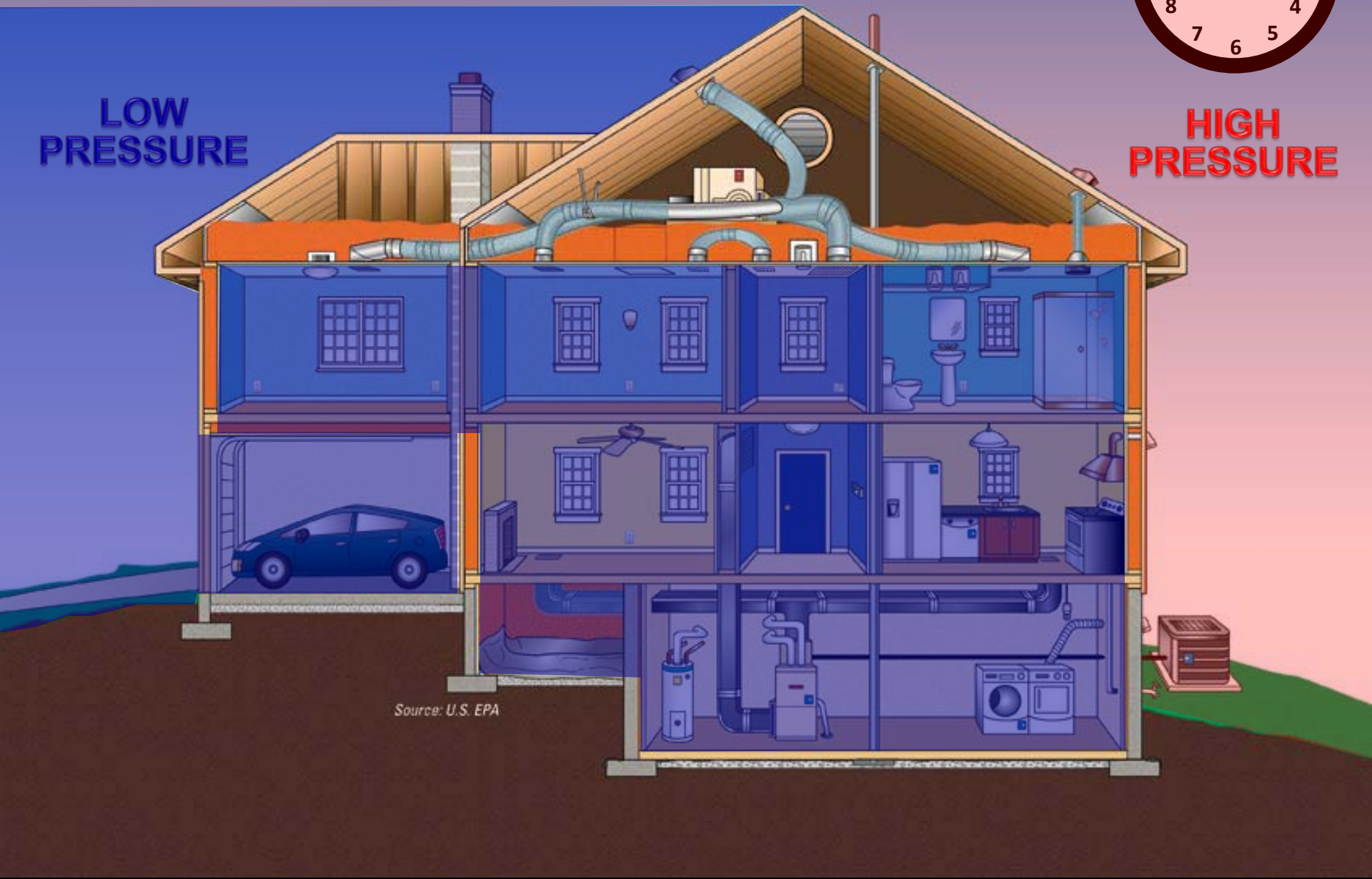
(Air Leakage Rate of $3ACH_{50}$ or $5ACH_{50}$ Required)

Use of a calibrated fan to quantify/evaluate air leakage…

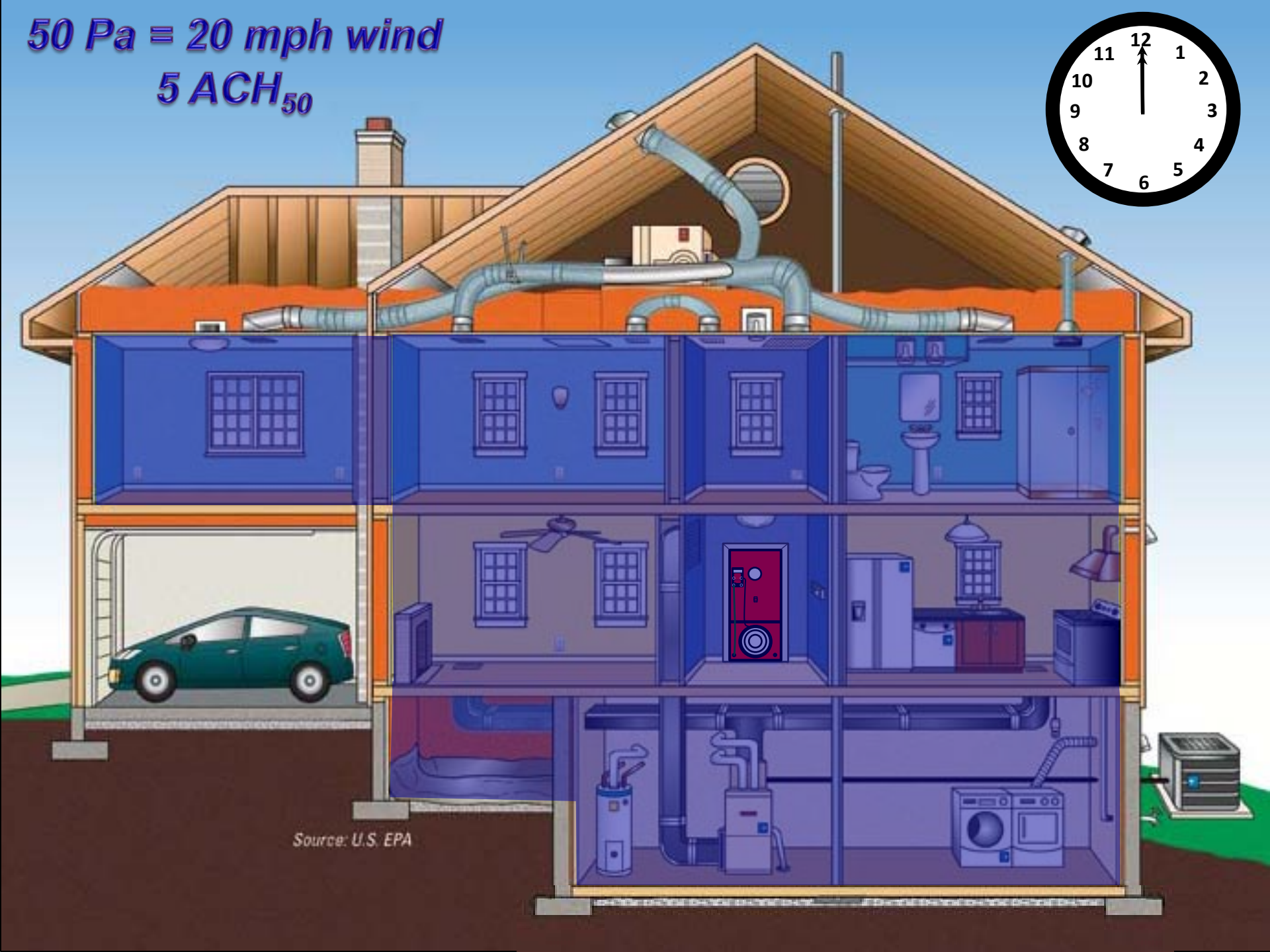
- **Installation**. Components of the envelope and air barrier are installed in accordance with the thermal- and air-barrier table.
- **Testing**. The building shall be tested/verified with a blower door to quantify rate of air leakage not exceeding 5 ACH50.
- Where required by the AHJ, Blower Door Testing shall be conducted by an ***approved*** third party.

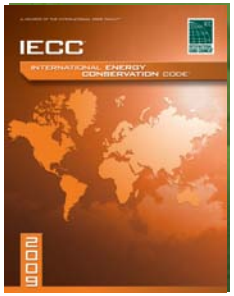
50 Pa = 20 mph wind
1 ACH_{50}

WIND PRESSURE



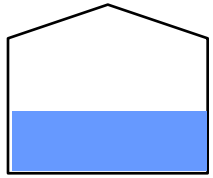
50 Pa = 20 mph wind
5 ACH_{50}



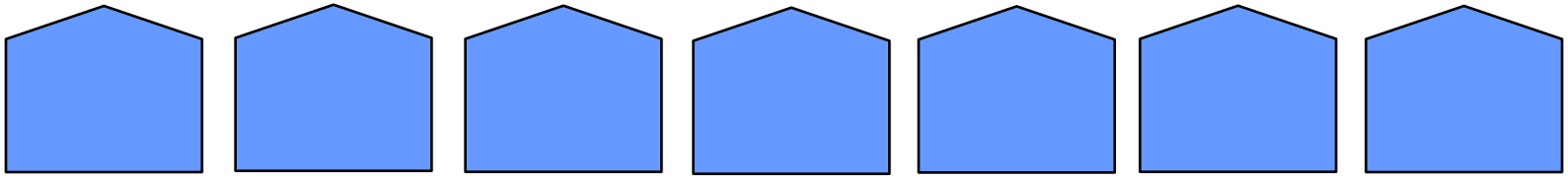


2009 IECC Air Leakage 7ACH$_{50}$

4 mph wind = 0.35 ACH$_{nat}$
(140 cfm or 8,400 cubic/ft of natural ventilation/hr)



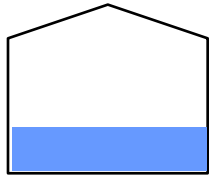
20 mph wind = 7.0 ACH$_{50}$
(2,800 cfm or 168,000 cubic/ft of natural ventilation/hr)



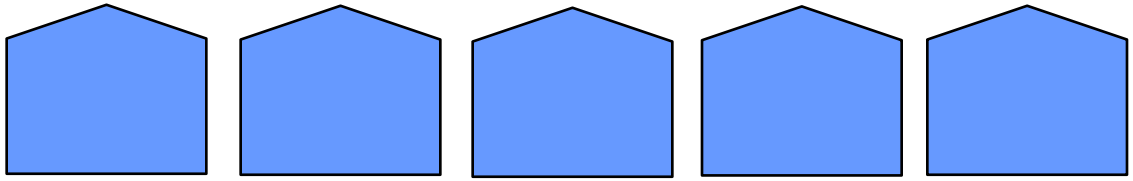
(1,500 sq/ft home w/full basement)

2012 IECC Air Leakage $5ACH_{50}$

4 mph wind = 0.25 ACH_{nat}
(100 cfm or 6,000 cubic/ft of natural ventilation/hr)



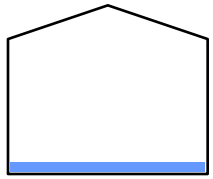
20 mph wind = 5.0 ACH_{50}
(2,000 cfm or 120,000 cubic/ft of natural ventilation/hr)



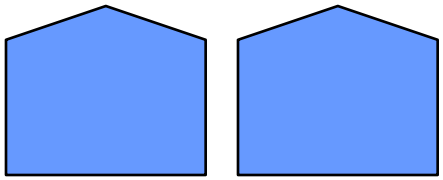
(1,500 sq/ft home w/full basement)

ENERGYSTAR Tighter than Code Air Leakage $2ACH_{50}$

4 mph wind = 0.1 ACH_{nat}
(40 cfm or 2,400 cubic/ft of natural ventilation/hr)



20 mph wind = 2.0 ACH_{50}
(800 cfm or 48,000 cubic/ft of natural ventilation/hr)



(1,500 sq/ft home w/full basement)

Duct Pressure Testing

2012 IECC "Total" Duct Leakage

(page R34 left side of page)

Total duct leakage is based on 3 cfm/100 ft^2 of CFA with no air handler, or 4 cfm/100 ft^2 with air handler installed.

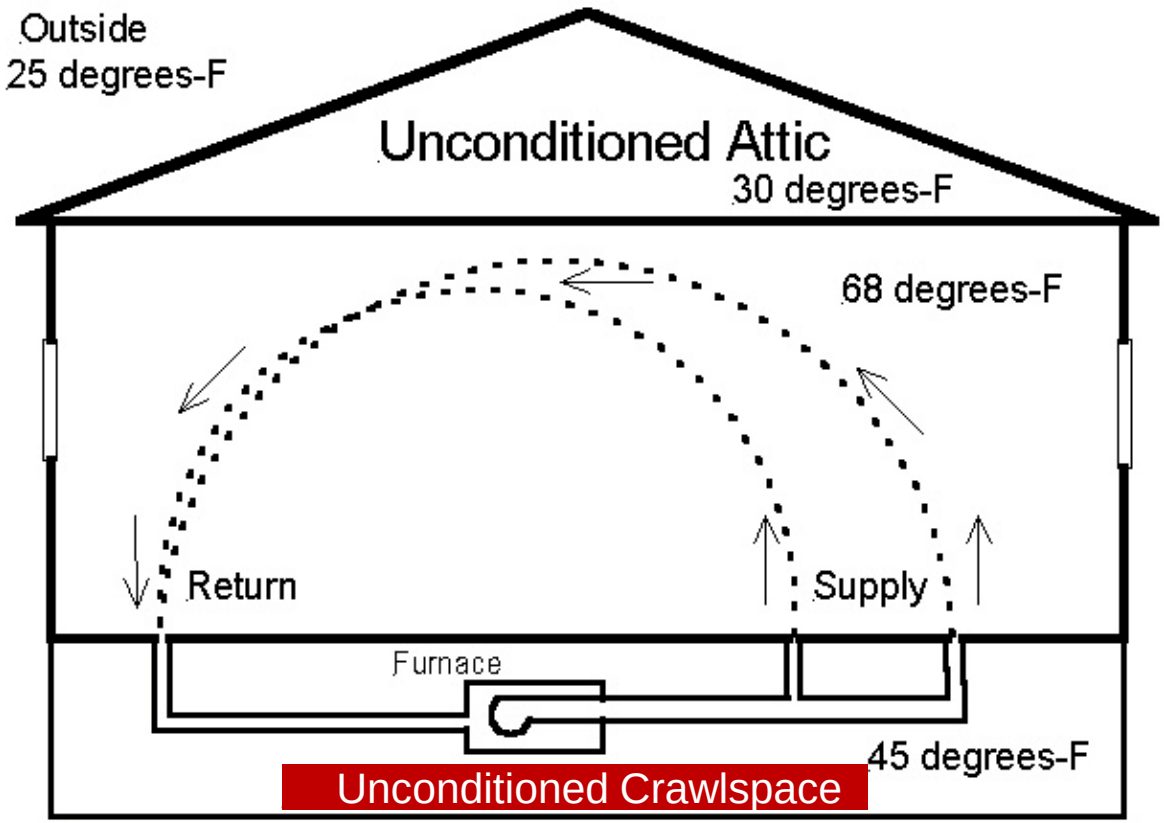
- Rough-in no air handler …… 3 cfm
- Rough-in with air handler ……. 4 cfm
- Post construction with air handler ….. 4 cfm

Example:

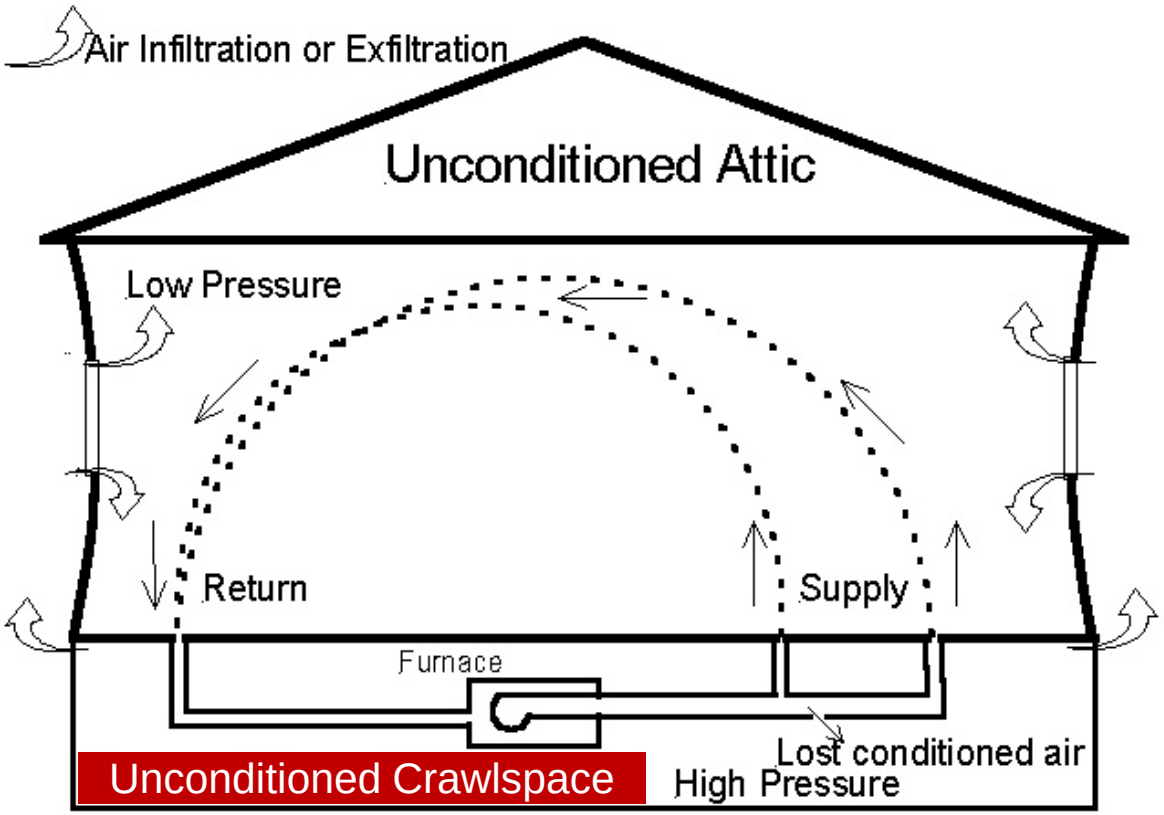
*(2,400 sq' ÷ 100) x 3 cfm = **72 cfm** of total duct leakage*

*(2,400 sq' ÷ 100) x 4 cfm = **96 cfm** of total duct leakage*

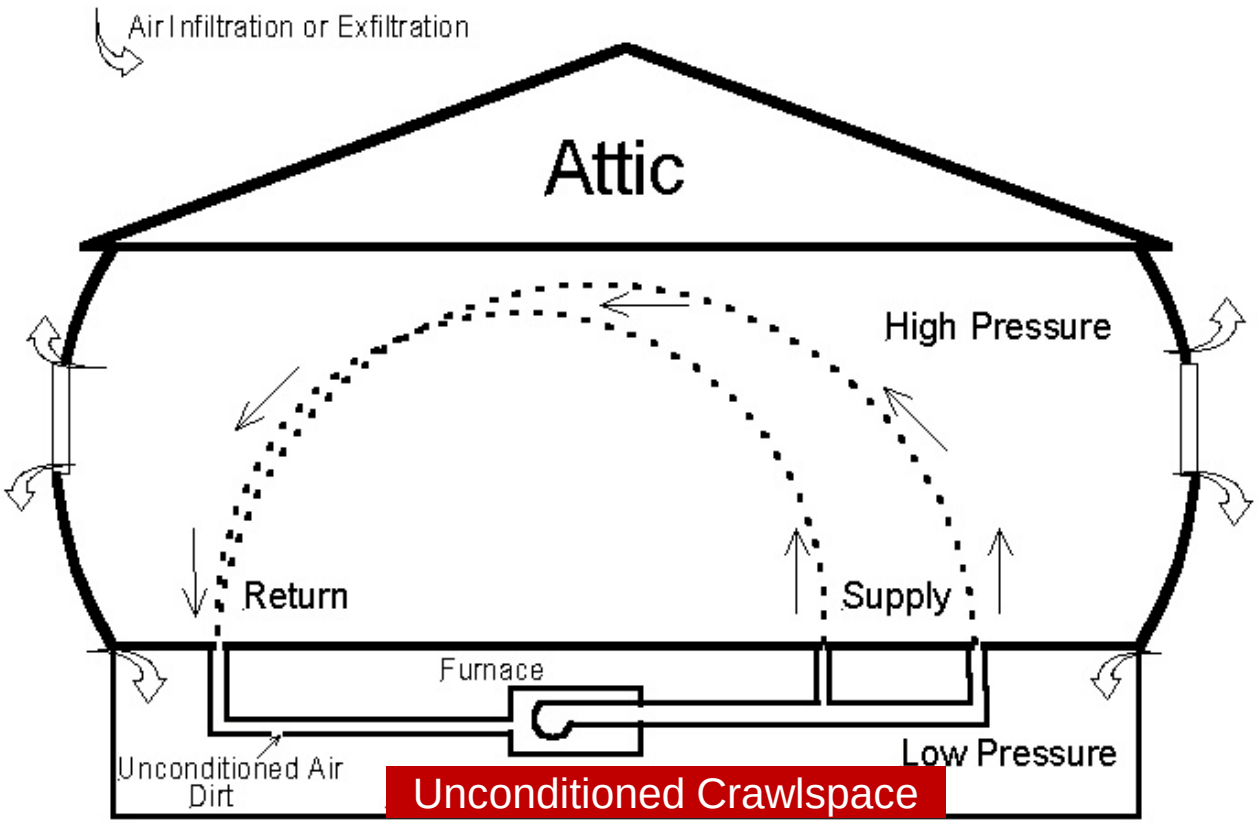
Balanced Duct System - No Leaks



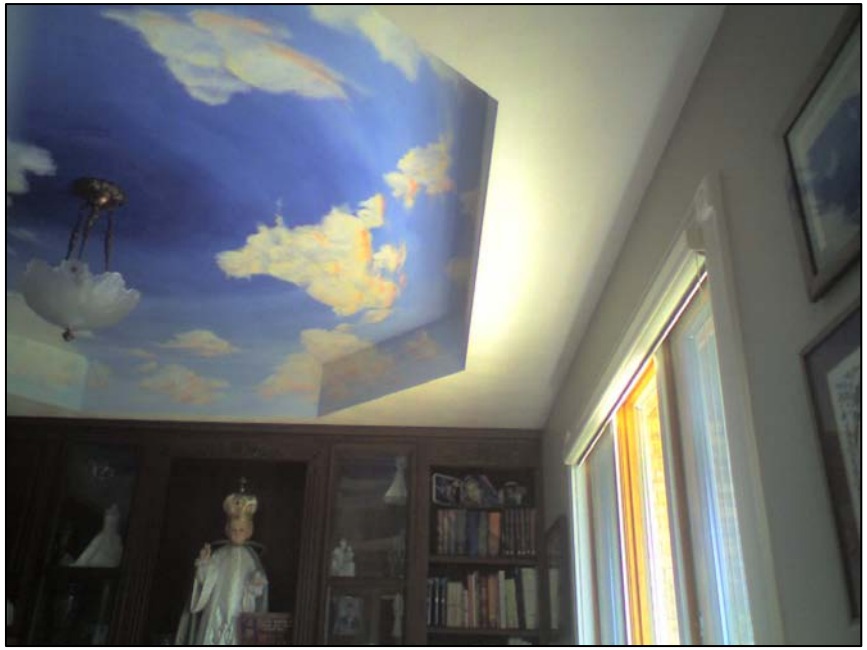
Supply Leaks in Unconditioned Space



Return Leaks in Unconditioned Space



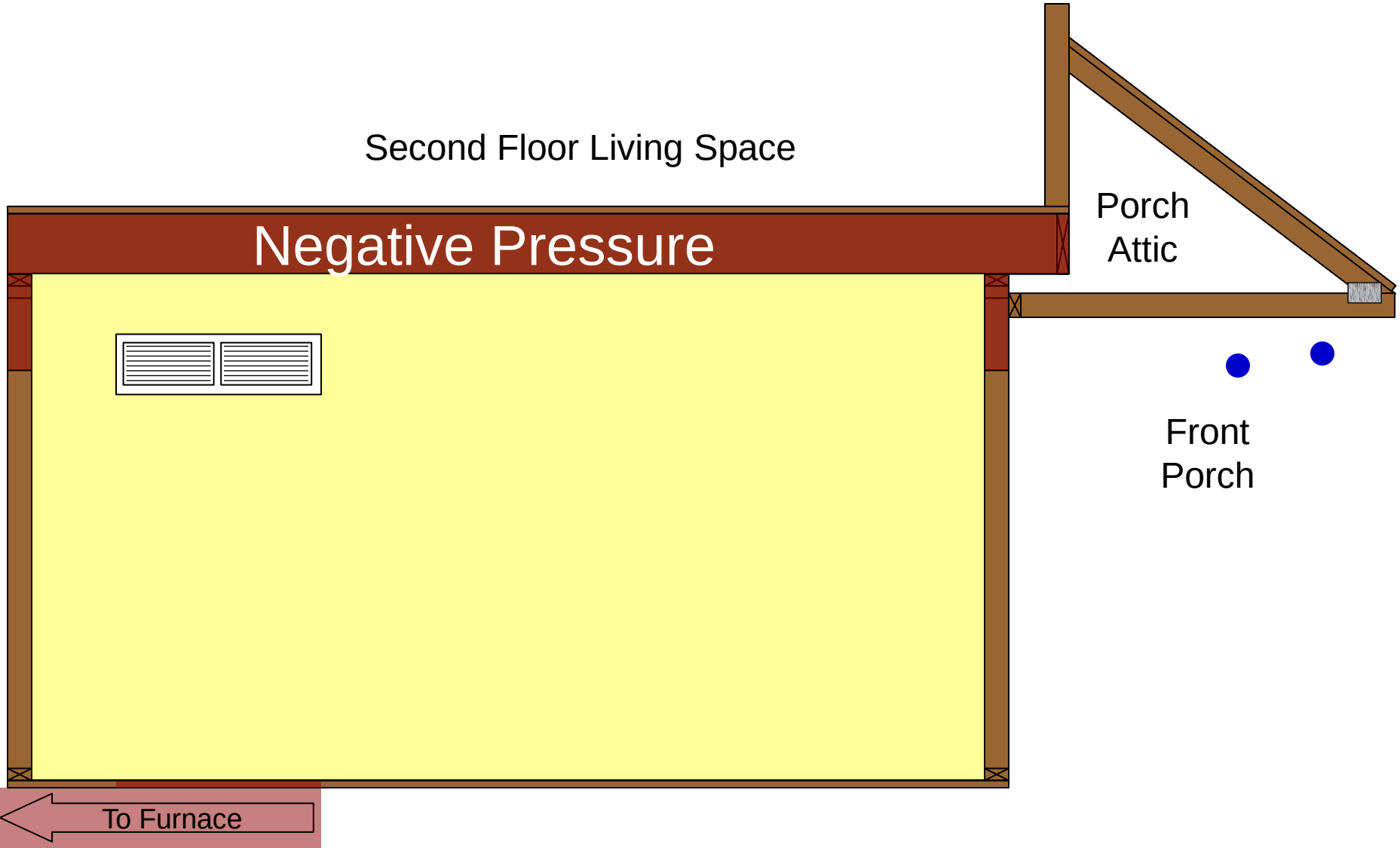
Tray Ceiling - Return Duct



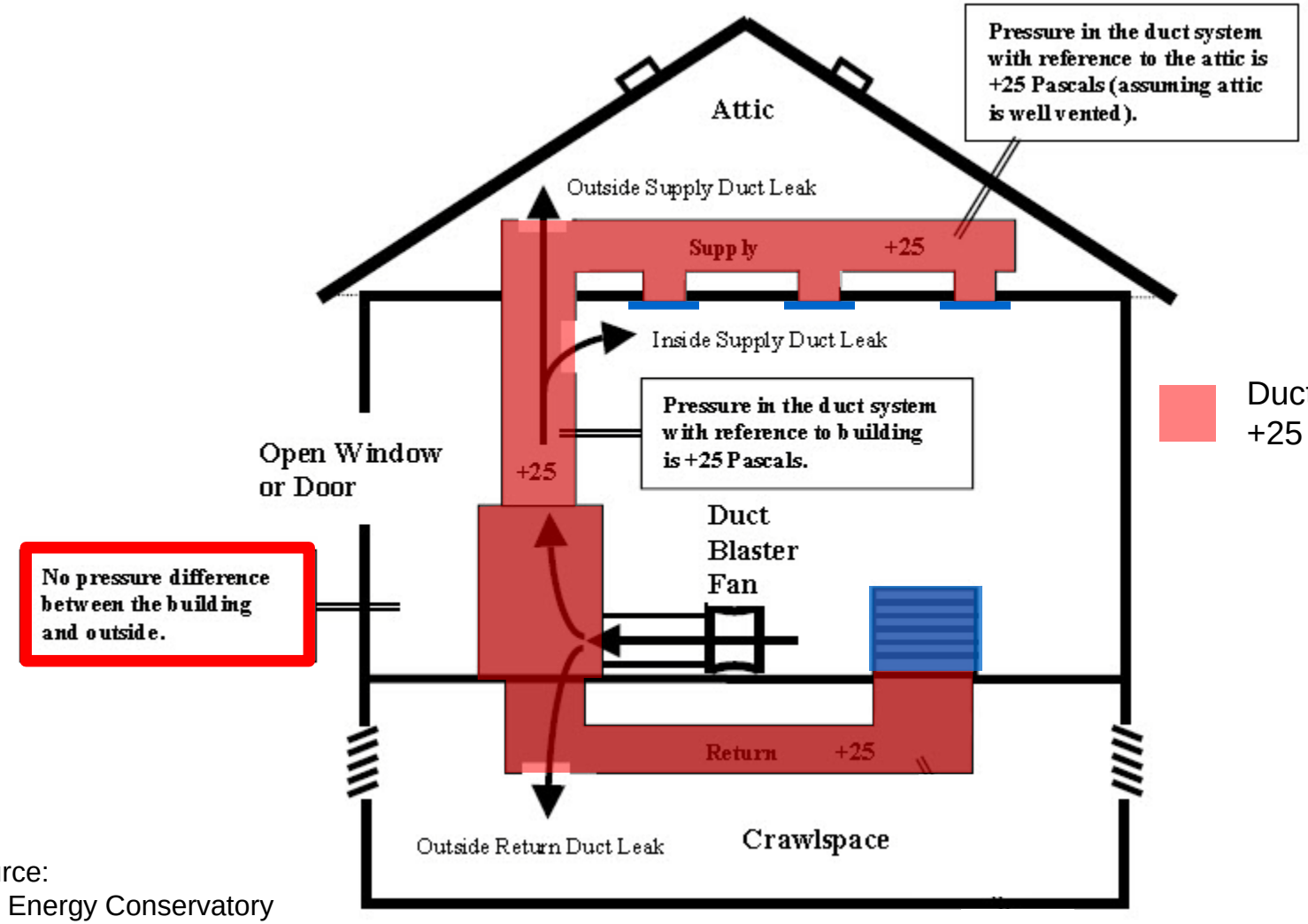
Infrared photos was taken without the blower door operating!!

Building Cavities are Not Ducts

Duct Leakage to the Outside

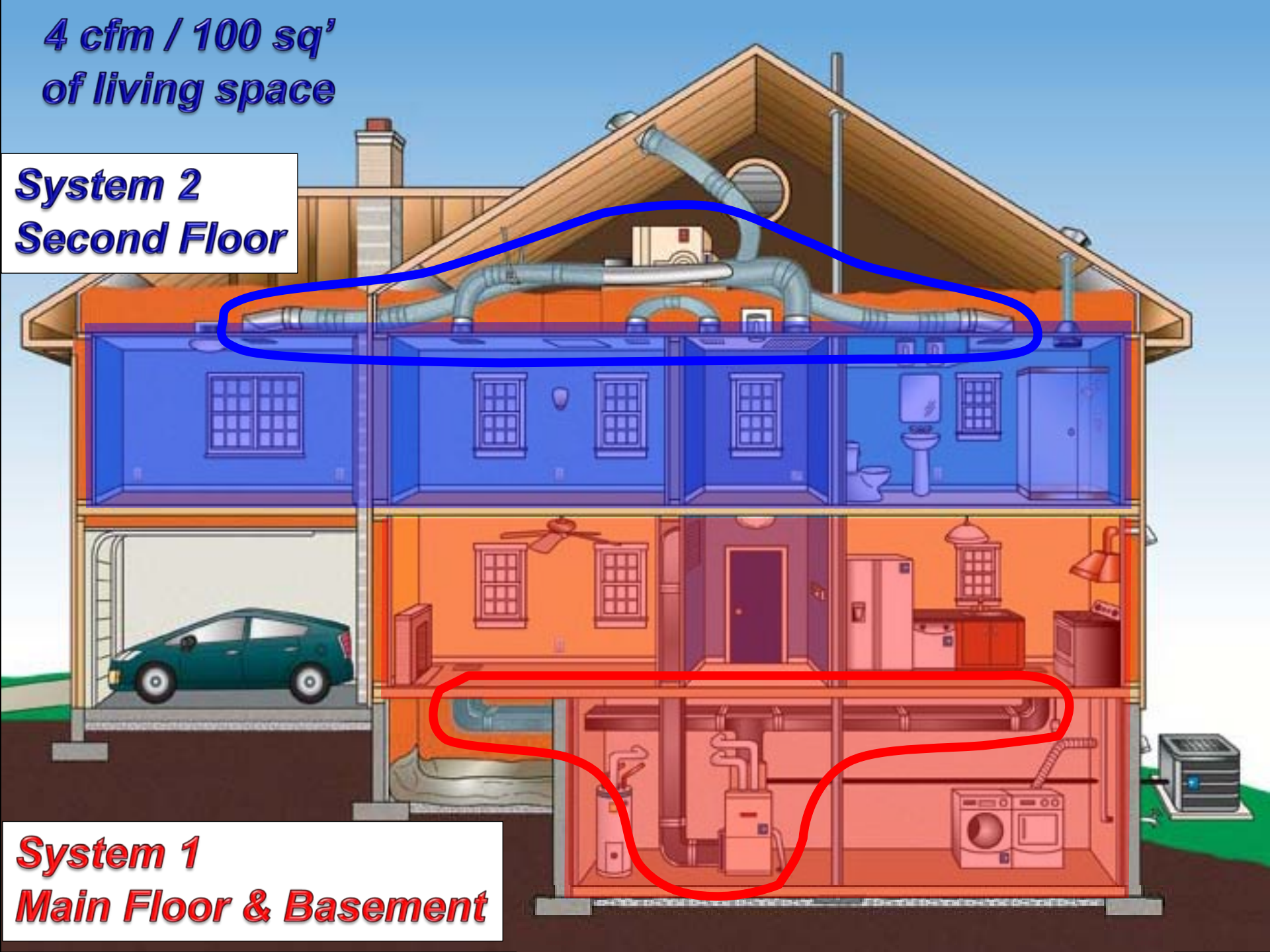


"Total" Duct Leakage Test
(no Blower Door)

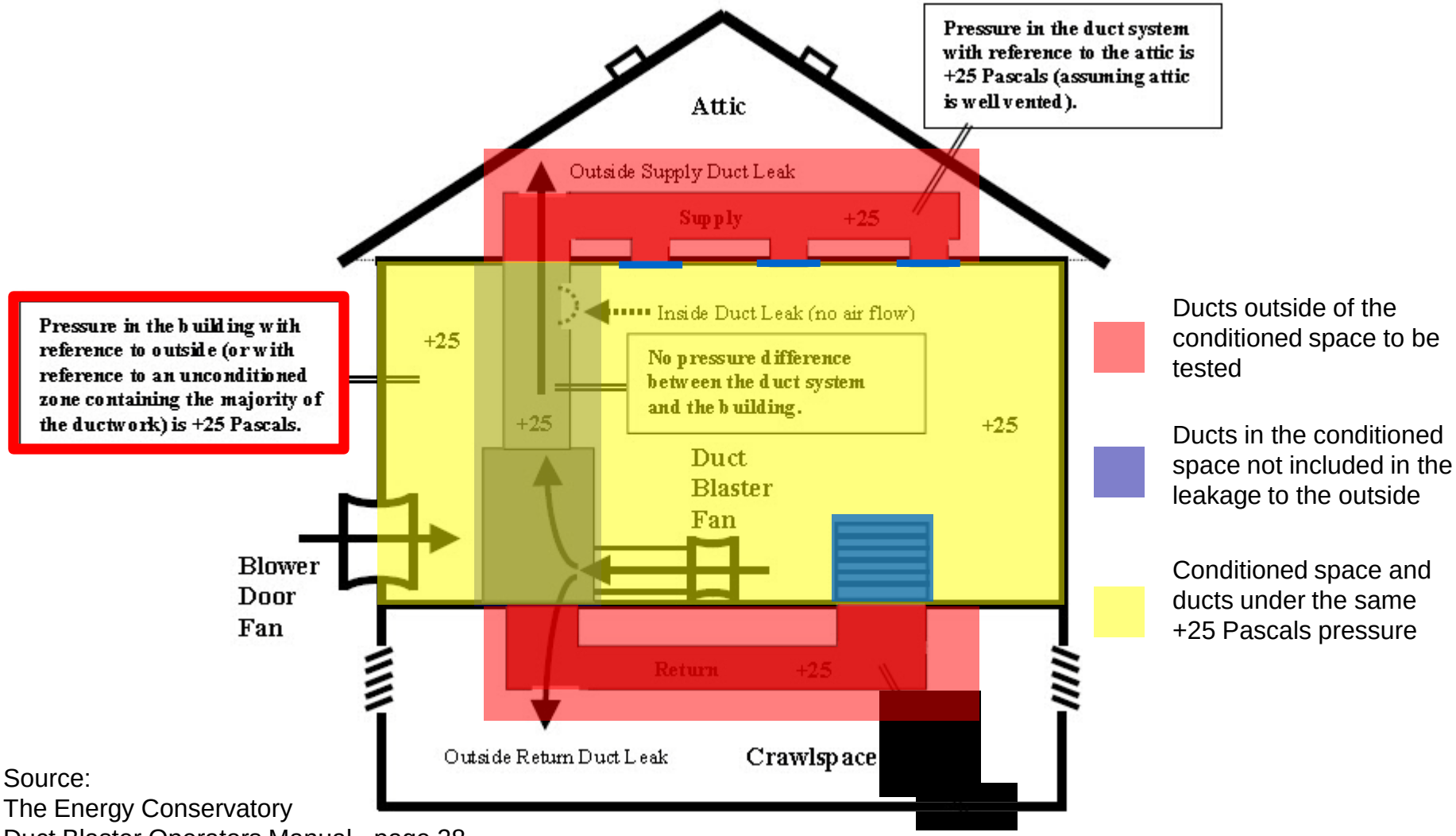


Ducts pressurized to +25 Pascals

Source:
The Energy Conservatory
Duct Blaster Operators Manual - page 18


4 cfm / 100 sq'
of living space
System 2
Second Floor
System 1
Main Floor & Basement

Duct Leakage to the "*Outside Only*" *(2009 IECC)*



Source:
The Energy Conservatory
Duct Blaster Operators Manual - page 28

Wrong- vs. Right-sized Equipment

Oversized equipment will:

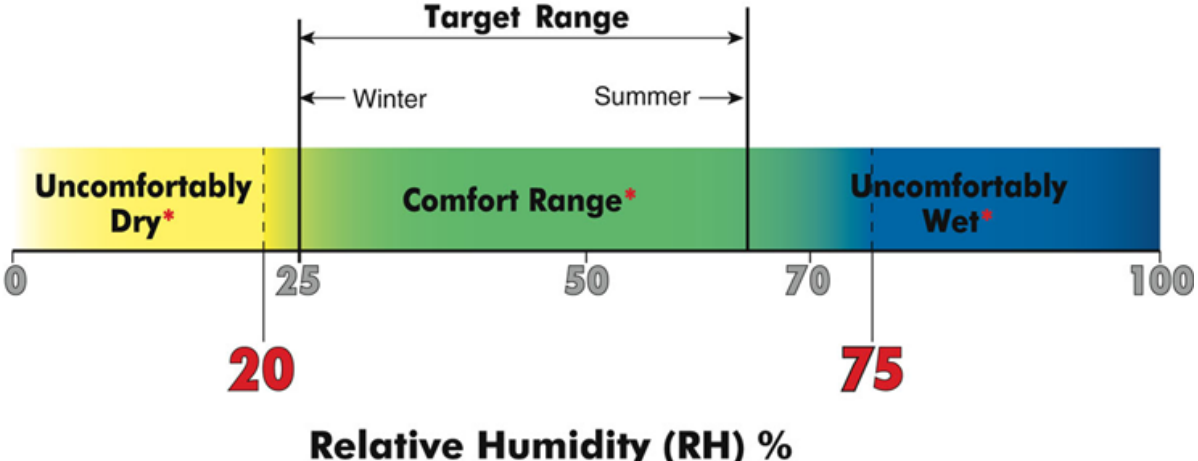
- Cause equipment to short-cycle; more frequent start-stops increase maintenance ($).
- Degrade summer humidity control.
- Increase potential for mold growth.
- Satisfy thermostat location but room hot/cold spots created.
- Require larger ducts, increased circuit sizing, larger refrigerant line sets ($).

Reasons for Oversized equipment:

- (1) A guess was made on the equipment's capacity at design conditions or
- (2) Mistakes were made in the equipment selection process.
- (3) Multi-zone systems add to installed cost.
- Great care is taken to measure and test equipment performance at operating conditions.
- Contractors using this data to select equipment will meet the comfort needs of their clients.

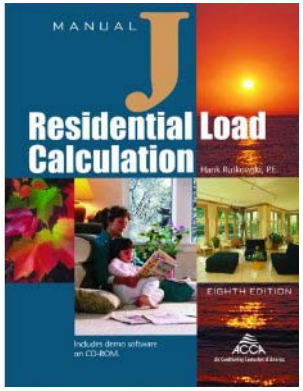
Undersized Equipment

- Does not maintain desired temperature and humidity. Equipment runs and runs but cannot satisfy the thermostat or the occupant.
- However, it is the lesser of the two evils:
  - Slightly undersized cooling equipment (5% or less) may actually provide more comfort at a lower cost.
  - You still get very good dehumidification on mild rainy days when you still want some cooling, and everyday you are running the A/C you are saving money.
- Small, modulating condensing gas furnaces should be available in 12-24 kBtuh capacities. Purely a manufacturing issue



R403.6 – Equipment Sizing ACCA Manual ‘J’, Manual ‘S’

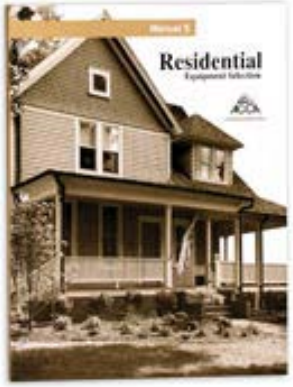
- *Manual J8th* is only used to calculate the heating and cooling loads.
- *Manual J8th* guides HVAC designers to use ACCA *Manual S* to select equipment that is the right size (see § 10-4 of *Manual S*).
- *Manual S* sets equipment sizing limits, as summarized in Table 1.



Manual S Equipment Selection Sizing Limitations		
Equipment	Sizing Limits	Reference
Furnaces	100% - 140% of total heating load	Section 2-2
Boilers	100% - 140% of total heating load	Section 2-2
Air conditioners	115% of total cooling load*	Section 3-4
Heat pumps	115%[1] or 125%[2] of total cooling load*	Section 4-4
Supplemental heat (heat pumps)		
• Electric	Based on equipment balance point	Section 4-8
• Dual fuel	100% - 140% of total heating load	Section 6-8
Emergency Heat (heat pumps)	Based on local codes	Section 4-9

Manual S Input for Design Air Flow (*Manual D*)		
Mode of Operation	Requirement	Reference
• Heating	Temperature rise requirement	Section 2-6
• Cooling	Air flow associated with the selected equipment's capacity	Section 3-11

[1] Heat pumps in a *cooling* dominant climate are allowed to be 115% of the cooling load.
[2] Heat pumps in a *heating* dominant climate are allowed to be 125% of the cooling load.
* The size of the cooling equipment must be based on the same temperature and humidity conditions that were used to calculate the *Manual J* loads.



R403.6 – Equipment Sizing
Applying Manual S to a Heating Example

- Select a furnace for a home with a **57,000 Btu/h** heating requirement.
- Furnace must deliver as least **57,000 Btu/h** to maintain 70F in the home when the outdoor temperature dips to design temp.
- *Manual S* sets a sizing limit for furnaces using Table 1.
- Based on home's load and sizing limitations, the furnace must produce **57,000 Btu/h ≤ heating requirement ≤ 79,800 Btu/h**.
- Furnace shall have a capacity no larger than 140% of heating load.
  - (140% x 57,000 = 79,800 Btu/h)

R403.6 – Equipment Sizing
Applying Manual S to a Cooling Example

- Select an air conditioner for a home with the following *Manual J8th* cooling loads:

Sensible Cooling	22,000 Btu/h
Latent Cooling	8,000 Btu/h
Total Cooling	**30,000 Btu/h**

- *Manual S* sets a sizing limit for air conditioners using Table 1.
- Based on home's load and sizing limitations, air conditioner must produce **30,000 Btu/h ≤ cooling requirement ≤ 34,500 Btu/h**.
- Air conditioner shall have a capacity no larger than 115% of total cooling load.
  - (115% x 30,000 = 34,500)

Geothermal System Install/Pathology

1,650 ft^2, Single-Family Ranch on Crawl,

Marissa, IL (Climate Zone 4)

- Ceiling, R-38 (cellulose)
- Walls (2x4), R-13
- Windows, U-0.57(insulated wd)
  - 19% Glazing Area
  - 17% of Conditioned Floor Area
- 2x Doors, U-0.39
- Floor over crawl, R-30

Geothermal System Install

1,650 ft², Single-Family Ranch on Crawl,

Marissa, IL (Climate Zone 4)

Product specifications

MODEL	CAPACITY	HEATING		COOLING	
		BTU/HR	COP	BTU/HR	EER
GXT024C	Full Load	19,700	4.7	27,500	21.9
	Part Load	16,400	5.0	21,200	29.8
GXT036C	Full Load	28,600	4.4	39,300	19.8
	Part Load	22,600	4.4	29,600	28.7
GXT048C	Full Load	39,500	4.2	49,900	18.5
	Part Load	31,200	4.8	38,800	26.6
GXT060B	Full Load	47,400	3.9	60,100	17.1
	Part Load	37,300	4.4	47,200	24.0
GXT072B	Full Load	57,000	3.7	65,400	15.0
	Part Load	47,100	4.2	54,500	20.7

Unit Notes

Rated in accordance with ISO Standard 13256-1, which includes pump penalties.
Heating capacities based on 68.0°F DB, 59.0°F WB entering air temperature.
Cooling capacities based on 80.6°F DB, 66.2°F WB entering air temperature.
Entering water temperatures Full Load: 32°F heating / 77°F cooling.
Entering water temperatures Part Load: 41°F heating / 68°F cooling.

Right-J Analysis After Install

(Entire House)

Design Information

	Htg	Clg
Outside db (°F)	0	93
Inside db (°F)	68	70
Design TD (°F)	68	23
Daily range	-	M
Inside humidity (%)	30	50
Moisture difference (gr/lb)	27	57

Infiltration

Method	Simplified
Construction quality	Average
Fireplaces	1 (Average)

HEATING EQUIPMENT

Make	American Standard
Trade	AMERICAN STANDARD
Model	ADX2B060A9362A*
GAMA ID	2016732

Efficiency	92.1 AFUE	
Heating input	60000	Btuh
Heating output	55000	Btuh
Temperature rise	38	°F
Actual air flow	1350	cfm
Air flow factor	0.028	cfm/Btuh
Static pressure	0	in H2O
Space thermostat		

COOLING EQUIPMENT

Make	American Standard
Trade	ADP
Cond	4A7B3042A1
Coil	H,TE43(6,9)60+*UD1D140A9H5
ARI ref no.	3797672

Efficiency	11.5 EER, 14 SEER	
Sensible cooling	28350	Btuh
Latent cooling	12150	Btuh
Total cooling	40500	Btuh
Actual air flow	1350	cfm
Air flow factor	0.050	cfm/Btuh
Static pressure	0	in H2O
Load sensible heat ratio	0.85	

ROOM NAME	Area (ft²)	Htg load (Btuh)	Clg load (Btuh)	Htg AVF (cfm)	Clg AVF (cfm)
OFFICE	120	4577	1881	127	94
FOYER	74	2640	1179	73	59
BR 1	132	5014	2011	139	100
BATH	66	1216	397	34	20
KITCHEN	168	3095	1656	86	83
BREAKFAST NOOK	138	8391	6623	233	330
GREAT ROOM	488	7666	4006	213	200
WIC	66	1304	421	36	21
MASTER BR	208	9216	6305	256	315
MA					98
LA					30
					1360
Entire House					
Other equip loads		0	0		
Equip: @ 0.08 BSM			26431		
Latent cooling					
TOTALS	1651	48639	31389	1350	1350

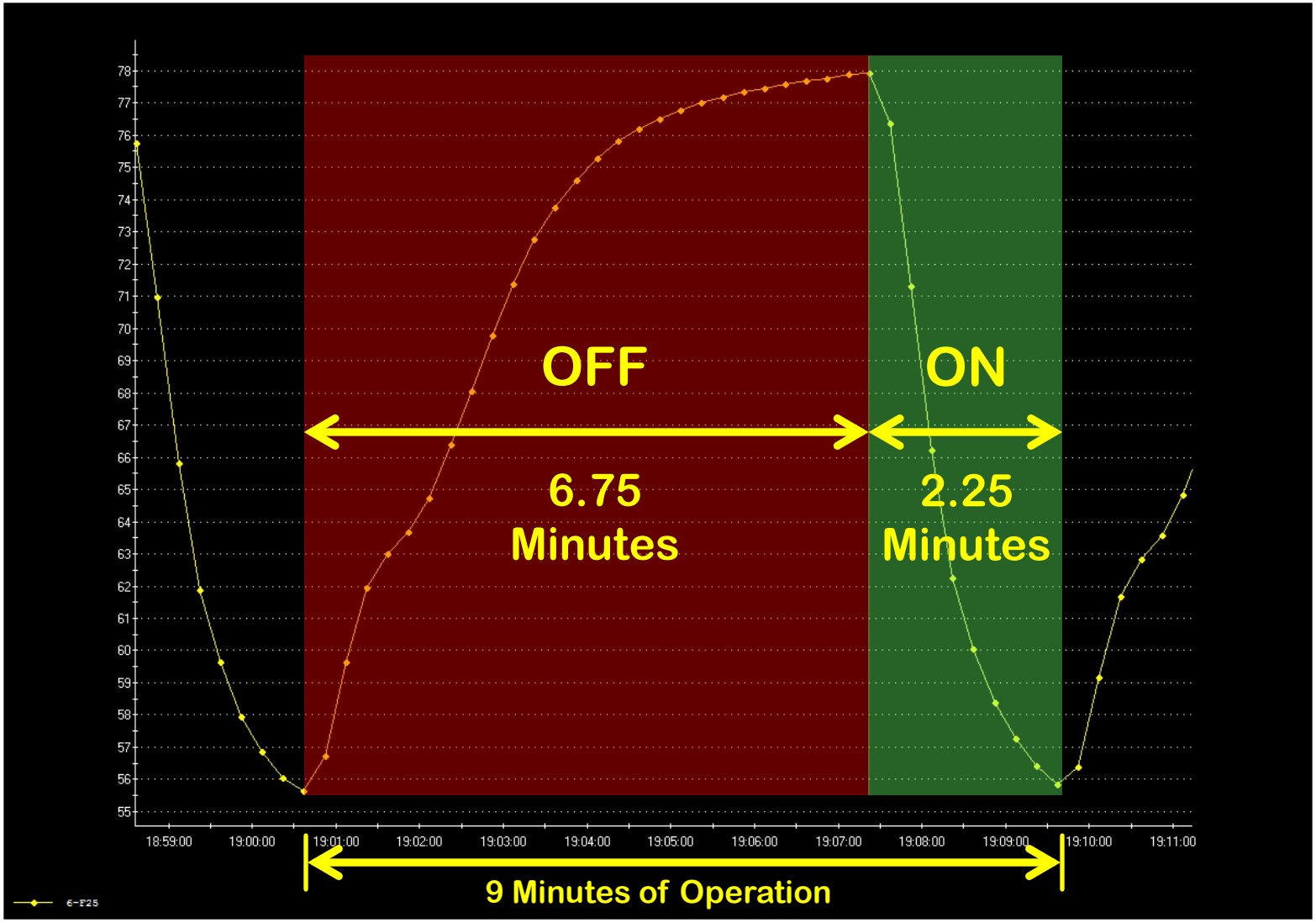
GXT036C					
	Full Load	28,600	4.4	39,300	19.8
	Part Load	22,600	4.4	29,600	28.7

2 Story Office Building

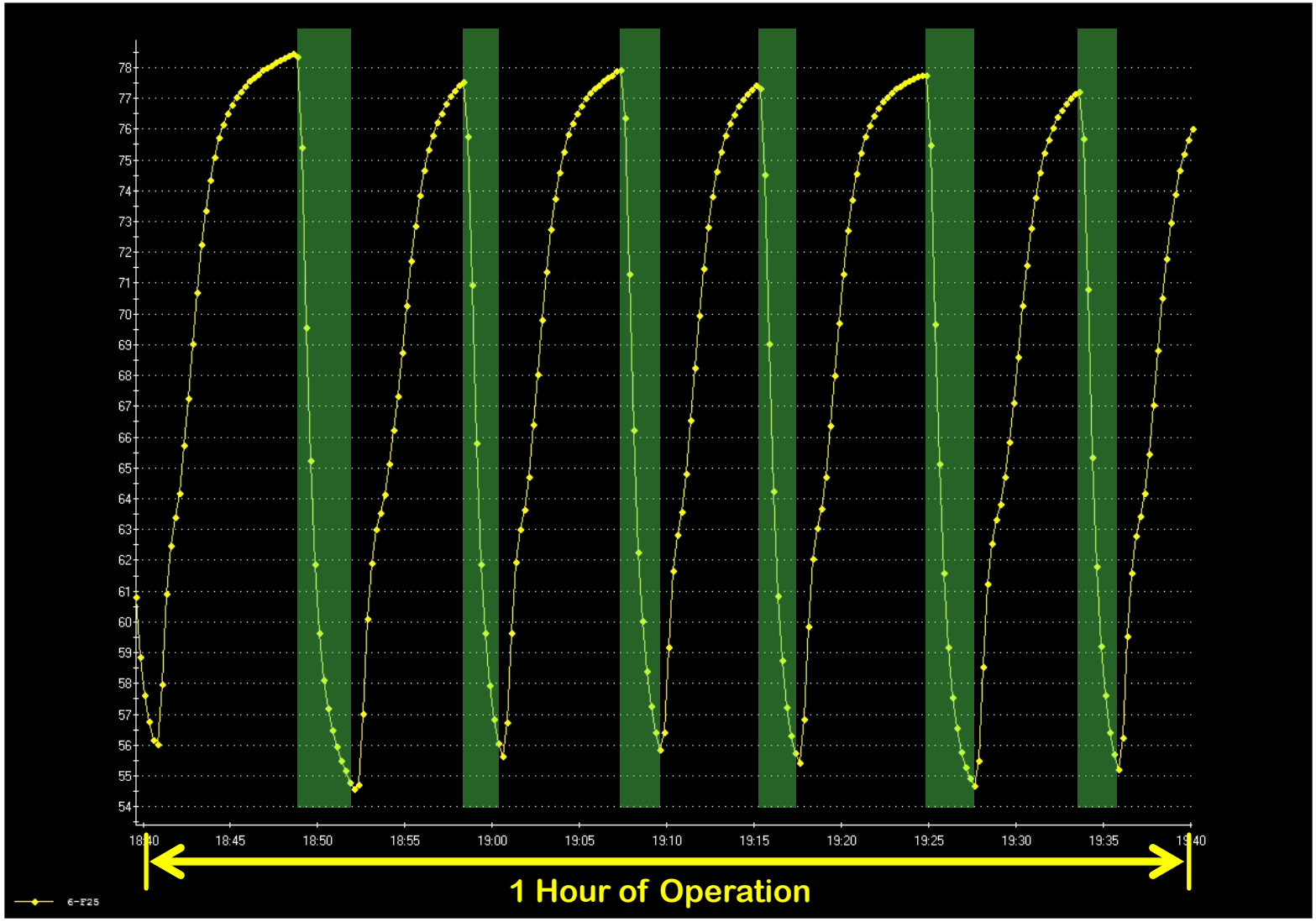
(with an efficient building shell)

13 condensers

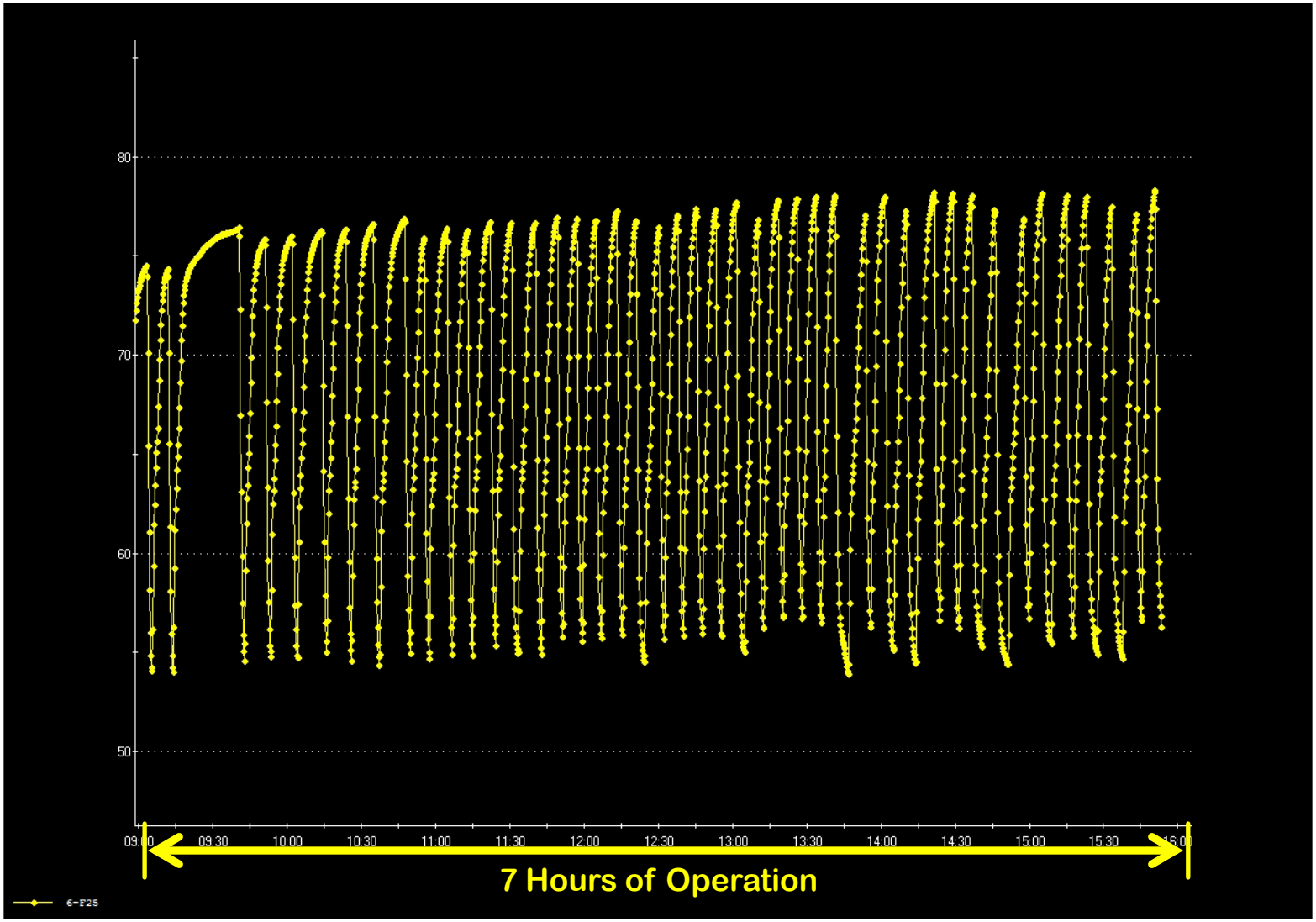
9 Minute Cycle Time (95° day)



1 Hour of Operation



7 Hours of Operation



Thank you!

Questions?

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