

## Right-Sized HVAC Design

*The Missing Ingredient in Most Homes*

Great Plains Energy Codes Conference  
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2012 INTERNATIONAL RESIDENTIAL ENERGY CONSERVATION CODE (IRC)  
2012 INTERNATIONAL ENERGY CONSERVATION CODE (IECC)  
2012 IECC Ventilation and Acceptable Indoor Air Quality in Low-Rise Residential Buildings

Darren Meyers, P.E., CEM, BPI-BA  
International Energy Conservation Consultants  
Education, Energy Engineering, Building Diagnostics, Sustainability  
[dmevers@ieccode.com](mailto:dmevers@ieccode.com)  
[www.ieccode.com](http://www.ieccode.com)

## The "House is a System"

### The "House is a System"

- HVAC are subsystems;
- HV & AC "condition air" for human comfort, duh!
- HVAC move and circulate air;
- HVAC affects building pressure;
- HVAC sustains life in the home (i.e., makes the house come alive)

### If HVAC is the backbone of a "House as a System," then:

- The load calculation is the backbone of the HVAC system
- Required by Code (IRC, IECC)
- Necessary to:
  - Select HVAC equipment size;
  - Determine supply CFM by room
  - Select supply outlets;
  - Select return inlets;
  - Design the duct system
  - Diagnose client comfort issues

## Standards of the Industry

**Load Calcs**

**System Size**

**Duct Design**

**ACCA**

## Why Manual J, S, D Training? What's Changed?

- For most in the Great Plains, 2009 IECC in force.
  - Blower door test (7ACH50) is optional to visual inspection,
  - Duct pressure testing (8 cfm/100 ft<sup>2</sup> of CFA), and
- For Illinois, 2012 IECC effective, January 1, 2013.
  - Mandatory Blower door test (5ACH50),
  - All "hard-ducted" w/ duct pressure testing (4 cfm/100 ft<sup>2</sup> of CFA), and
  - Mandatory whole-house ventilation (ASHRAE 62.2)
- For the average Illinois home: (2-story, 3-bdrm, 2,200 ft<sup>2</sup> w/bsmt)
  - A -33% reduction in envelope air-leakage (155/105 CFMn)
  - A deliberate mechanical ventilation strategy (60/240 CFMn)
  - A net reduction of 250 to 400 CFMn across the building envelope.

## Why Manual J, S, D Training?

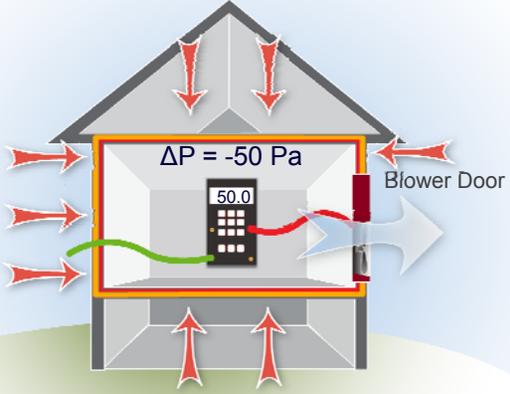
- Load estimating (MJ8), equipment sizing (MS), and duct-design (MD) concepts must be re-introduced to builders, contractors, and home owners.
- Why? They do not readily recognize that equipment size, and therefore installation and operating costs, dramatically decrease and long term comfort increases when load estimates are no longer based on rules-of-thumb and record-setting weather conditions.
- **NOTE:** *The amount of effort devoted to the design work is closely tied to the designer's ability to charge for this work.*



## How Air Moves Through Buildings

**Pressure (Wind)**    **Convection (stack effect)**    **Mechanical (fans)**

## Blower Door Testing



$\Delta P = -50 \text{ Pa}$

Blower Door

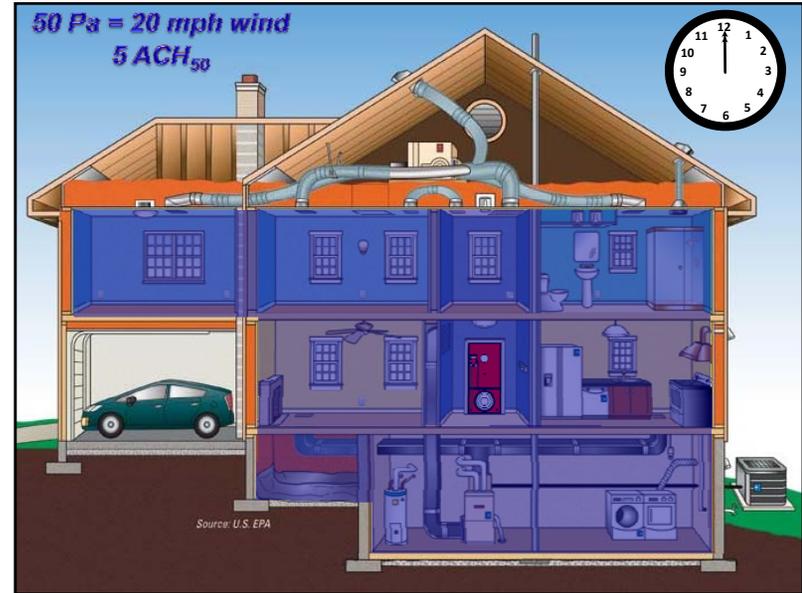
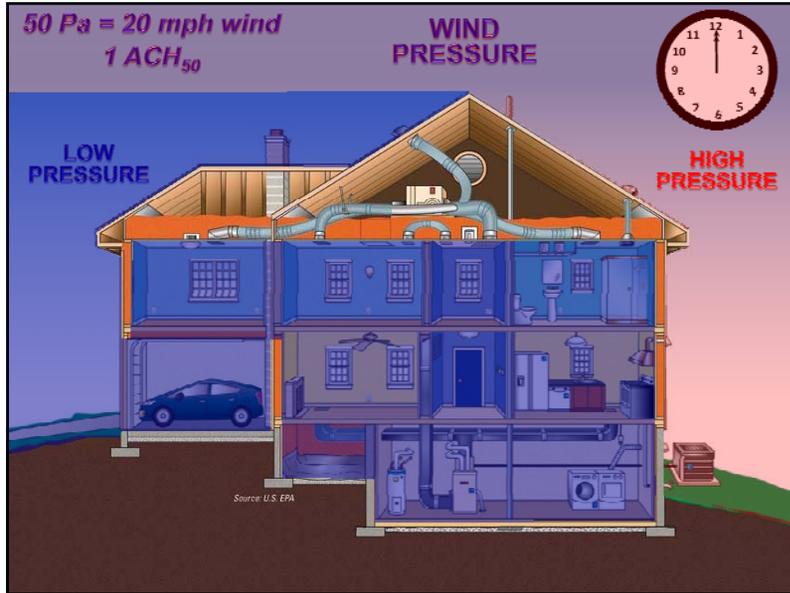
## R402.4.1.1 – Blower Door Testing

(Air Leakage Rate of 3ACH<sub>50</sub> or 5ACH<sub>50</sub> Required)

**Use of a calibrated fan to quantify/evaluate air leakage...**

- **Installation.** Components of the envelope and air barrier are installed in accordance with the thermal- and air-barrier table.
- **Testing.** The building shall be tested/verified with a blower door to quantify rate of air leakage not exceeding 5 ACH<sub>50</sub>.
- Where required by the AHJ, Blower Door Testing shall be conducted by an **approved** third party.





**Calculating ACH<sub>50</sub>**

**ACH<sub>50</sub> = (CFM<sub>50</sub> X 60 min/hour) ÷ Vol.**

- Blower Door Flow Reading = 2,550 cfm<sub>50</sub>
- House Volume = 27,000 cu/ft
- **ACH<sub>50</sub> = (2,550 x 60) ÷ 27,000 = 5.7 ACH<sub>50</sub>**  
*(Multiply by 60 to convert from minutes to hours)*

**Approximate Leakage Area**

Divide CFM<sub>50</sub> by 10  
 Then take Sq. Root  
*For example:*  
 2,550 CFM<sub>50</sub> ÷ 10 = 255 sq"  
 √ 255 = 16" x 16" hole

It's like having a window open 24 / 7 / 365

### 2009 IECC Air Leakage 7ACH<sub>50</sub>

4 mph wind = 0.35 ACH<sub>nat</sub>  
(140 cfm or 8,400 cubic/ft of natural ventilation/hr)

20 mph wind = 7.0 ACH<sub>50</sub>  
(2,800 cfm or 168,000 cubic/ft of natural ventilation/hr)

(1,500 sq/ft home w/full basement)

### 2012 IECC Air Leakage 5ACH<sub>50</sub>

4 mph wind = 0.25 ACH<sub>nat</sub>  
(100 cfm or 6,000 cubic/ft of natural ventilation/hr)

20 mph wind = 5.0 ACH<sub>50</sub>  
(2,000 cfm or 120,000 cubic/ft of natural ventilation/hr)

(1,500 sq/ft home w/full basement)

### ENERGYSTAR Tighter than Code Air Leakage 2ACH<sub>50</sub>

4 mph wind = 0.1 ACH<sub>nat</sub>  
(40 cfm or 2,400 cubic/ft of natural ventilation/hr)

20 mph wind = 2.0 ACH<sub>50</sub>  
(800 cfm or 48,000 cubic/ft of natural ventilation/hr)

(1,500 sq/ft home w/full basement)

### Mechanical vs Natural Ventilation

Based on a single-story 1,500 sq/ft house

(N-factor = 14.58)	ACH <sub>50</sub>	ACH <sub>n</sub>
CFM50 = 4,000	20.0	1.37
CFM50 = 2,000	10.0	.69
CFM50 = 1,000	5.5	.34
CFM50 = 1,000 + 75 CFM fan		

Source: University of Illinois – Building Research Council



### Return Leaks in Unconditioned Space

The diagram illustrates a cross-section of a house with a gabled roof. Below the main living area is an 'Unconditioned Crawlspace' containing a furnace. A return duct is shown with arrows indicating air being pulled from the crawlspace into the furnace. From the furnace, supply ducts lead to the attic. The attic is labeled 'Attic' and shows 'High Pressure' air being pushed into it. Arrows indicate air infiltration and exfiltration through the roof and walls. The furnace area is labeled 'Low Pressure'. The crawlspace contains 'Unconditioned Air Dirt'.

### Tray Ceiling - Return Duct

Two side-by-side infrared photos of a room with a tray ceiling. The left photo shows a temperature of 70.8 and the right photo shows a temperature of 63.0. A FLIR logo is visible in the bottom left corner of the left photo.

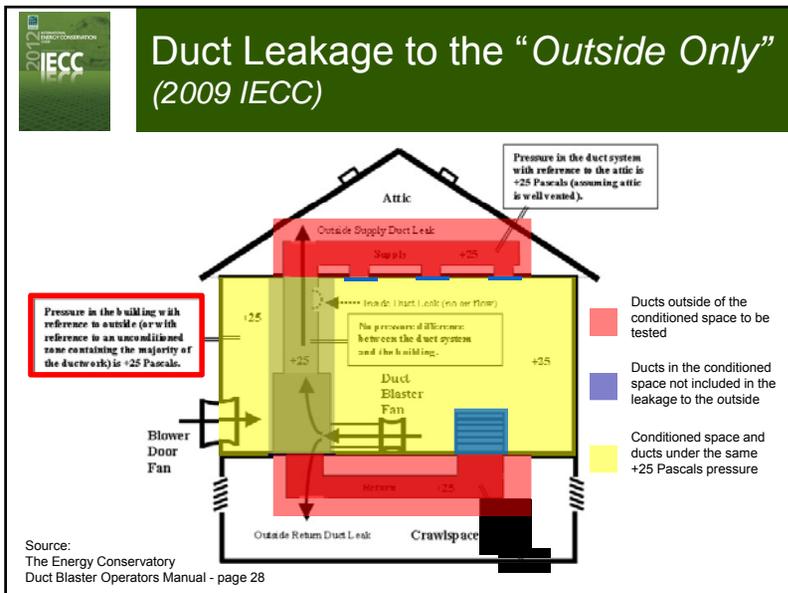
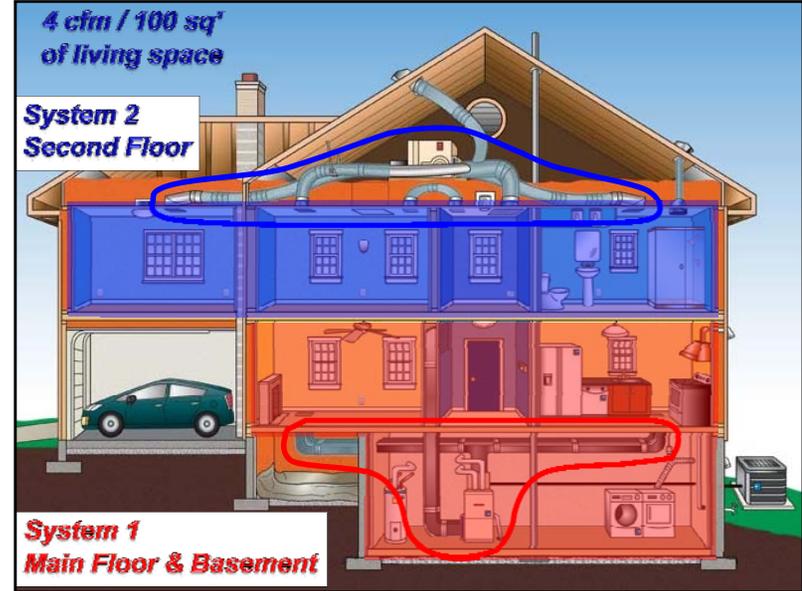
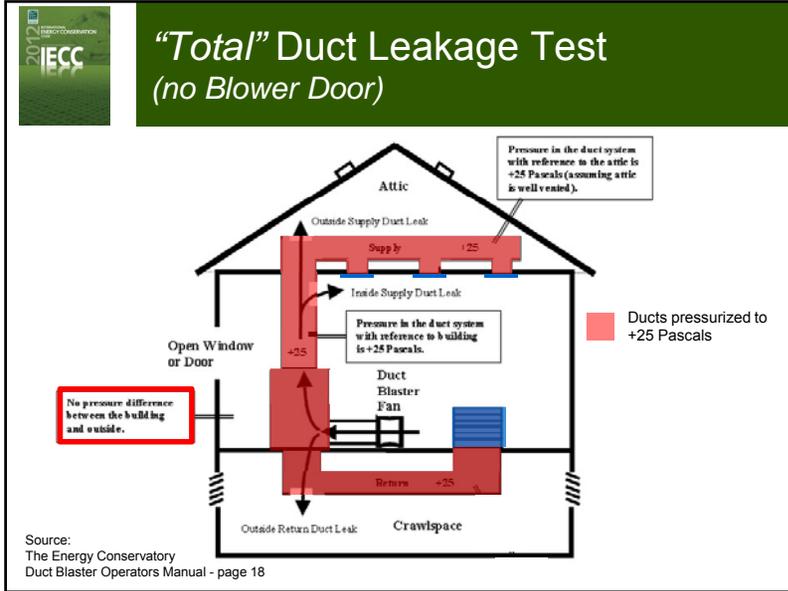
Infrared photos was taken without the blower door operating!!

### Building Cavities are Not Ducts

A photograph of a two-story house with a brick and siding exterior. Two yellow boxes highlight the window and door areas on the second floor.

### Duct Leakage to the Outside

A floor plan diagram of a second floor. The main living space is highlighted in yellow and labeled 'Second Floor Living Space'. A brown bar across the top of this space is labeled 'Negative Pressure'. To the right, there is a 'Porch Attic' and a 'Front Porch'. An arrow at the bottom left points 'To Furnace'.



### Wrong- vs. Right-sized Equipment

#### Oversized equipment will:

- Cause equipment to short-cycle; more frequent start-stops increase maintenance (\$).
- Degrade summer humidity control.
- Increase potential for mold growth.
- Satisfy thermostat location but room hot/cold spots created.
- Require larger ducts, increased circuit sizing, larger refrigerant line sets (\$).

#### Reasons for Oversized equipment:

- (1) A guess was made on the equipment’s capacity at design conditions or
- (2) Mistakes were made in the equipment selection process.
- (3) Multi-zone systems add to installed cost.
- Great care is taken to measure and test equipment performance at operating conditions.
- Contractors using this data to select equipment will meet the comfort needs of their clients.

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## 2012 IECC Undersized Equipment

- Does not maintain desired temperature and humidity. Equipment runs and runs but cannot satisfy the thermostat or the occupant.
- However, it is the lesser of the two evils:
  - Slightly undersized cooling equipment (5% or less) may actually provide more comfort at a lower cost.
  - You still get very good dehumidification on mild rainy days when you still want some cooling, and everyday you are running the A/C you are saving money.
- Small, modulating condensing gas furnaces should be available in 12-24 kBtuh capacities. Purely a manufacturing issue

Relative Humidity (RH) %

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## 2012 IECC Oversizing Impacts

Comfort	Equipment
<ul style="list-style-type: none"> <li>Marginal part load temperature control</li> <li>Large temperature differences between rooms</li> <li>Degraded humidity control</li> <li>Drafts and noise</li> <li>Occupant discomfort/dissatisfaction</li> </ul>	<ul style="list-style-type: none"> <li>Larger ducts installed</li> <li>Increased electrical circuit sizing</li> <li>Excessive part-load operation                             <ul style="list-style-type: none"> <li>Frequent cycling (loading/unloading)</li> <li>Shorter equipment life</li> </ul> </li> <li>Nuisance service calls</li> </ul>
Economic	Health
<ul style="list-style-type: none"> <li>Higher installed costs</li> <li>Increased operating expense</li> <li>Increased installed load on the public utility system</li> </ul>	<ul style="list-style-type: none"> <li>Potential to contribute to mold growth</li> <li>Potential to contribute to asthma and other respiratory conditions</li> </ul>

## 2012 IECC R403.6 – Equipment Sizing ACCA Manual ‘J’, Manual ‘S’

- Manual J*<sup>8th</sup> is only used to calculate the heating and cooling loads.
- Manual J*<sup>8th</sup> guides HVAC designers to use ACCA *Manual S* to select equipment that is the right size (see § 10-4 of *Manual S*).
- Manual S* sets equipment sizing limits, as summarized in Table 1.

Manual S Equipment Selection Sizing Limitations		
Equipment	Sizing Limits	Reference
Furnaces	100% - 140% of total heating load	Section 2-2
Boilers	100% - 140% of total heating load	Section 2-2
Air conditioners	115% of total cooling load*	Section 3-4
Heat pumps	115% <sup>1</sup> or 125% <sup>2</sup> of total cooling load*	Section 4-4
Supplemental heat (heat pumps)		
• Electric	Based on equipment balance point	Section 4-8
• Dual fuel	100% - 140% of total heating load	Section 6-8
Emergency Heat (heat pumps)	Based on local codes	Section 4-9
Manual S Input for Design Air Flow (Manual D)		
Mode of Operation	Requirement	Reference
• Heating	Temperature rise requirement	Section 2-6
• Cooling	Air flow associated with the selected equipment's capacity	Section 3-11

<sup>1</sup> Heat pumps in a cooling dominant climate are allowed to be 115% of the cooling load.  
<sup>2</sup> Heat pumps in a heating dominant climate are allowed to be 125% of the cooling load.  
 \* The size of the cooling equipment must be based on the same temperature and humidity conditions that were used to calculate the *Manual J* loads.

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## 2012 IECC R403.6 – Equipment Sizing Applying Manual S to a Heating Example

- Select a furnace for a home with a **57,000 Btu/h** heating requirement.
- Furnace must deliver at least **57,000 Btu/h** to maintain 70F in the home when the outdoor temperature dips to design temp.
- Manual S* sets a sizing limit for furnaces using Table 1.
- Based on home's load and sizing limitations, the furnace must produce **57,000 Btu/h ≤ heating requirement ≤ 79,800 Btu/h**.
- Furnace shall have a capacity no larger than 140% of heating load.
  - (140% x 57,000 = 79,800 Btu/h)

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## R403.6 – Equipment Sizing Applying Manual S to a Cooling Example

- Select an air conditioner for a home with the following *Manual J8<sup>th</sup>* cooling loads:
 

Sensible Cooling	22,000 Btu/h
Latent Cooling	<u>8,000 Btu/h</u>
<b>Total Cooling</b>	<b>30,000 Btu/h</b>
- *Manual S* sets a sizing limit for air conditioners using Table 1.
- Based on home's load and sizing limitations, air conditioner must produce **30,000 Btu/h ≤ cooling requirement ≤ 34,500 Btu/h**.
- Air conditioner shall have a capacity no larger than 115% of total cooling load.
  - (115% x 30,000 = 34,500)

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## Geothermal System Install/Pathology

*1,650 ft<sup>2</sup>, Single-Family Ranch on Crawl,  
Marissa, IL (Climate Zone 4)*

- Ceiling, R-38 (cellulose)
- Walls (2x4), R-13
- Windows, U-0.57 (insulated wd)
  - 19% Glazing Area
  - 17% of Conditioned Floor Area
- 2x Doors, U-0.39
- Floor over crawl, R-30



## Geothermal System Install

*1,650 ft<sup>2</sup>, Single-Family Ranch on Crawl,  
Marissa, IL (Climate Zone 4)*

Product specifications

MODEL	CAPACITY	HEATING		COOLING	
		BTU/HR	COP	BTU/HR	EER
GXT024C	Full Load Part Load	19,700 16,400	4.7 5.0	27,500 21,200	21.9 29.8
GXT036C	Full Load Part Load	28,600 22,600	4.4 4.4	39,300 29,600	19.8 20.7
GXT048C	Full Load Part Load	39,500 31,200	4.2 4.8	49,900 38,800	18.5 26.6
GXT060B	Full Load Part Load	47,400 37,300	3.9 4.4	60,100 47,200	17.1 24.0
GXT072B	Full Load Part Load	57,000 47,100	3.7 4.2	65,400 54,500	15.0 20.7



**Unit Notes**  
 Rated in accordance with ISO Standard 13256-1, which includes pump penalties.  
 Heating capacities based on 68.0°F DB, 59.0°F WB entering air temperature.  
 Cooling capacities based on 80.6°F DB, 66.2°F WB entering air temperature.  
 Entering water temperatures Full Load, 32°F heating / 77°F cooling.  
 Entering water temperatures Part Load, 41°F heating / 60°F cooling.

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## Right-J Analysis After Install

*(Entire House)*

Design Information					
	Htg	Cig	Method	Infiltration	Simplified
Outside db (°F)	68	93			
Inside db (°F)	68	70	Combustion quality		Average
Design TD (°F)	68	23	Fireplaces		1 (Average)
Daily range	30	50			
Inside humidity (%)	30	50			
Moisture difference (gr/lb)	27	67			

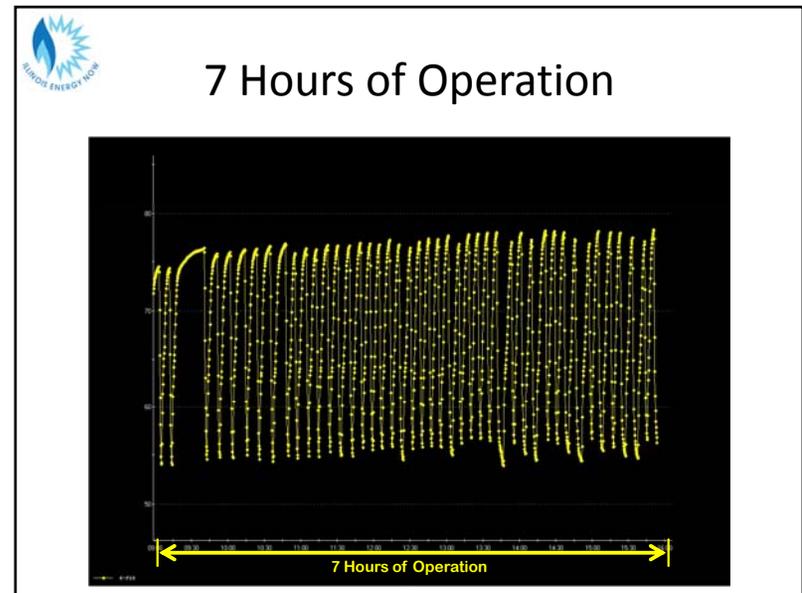
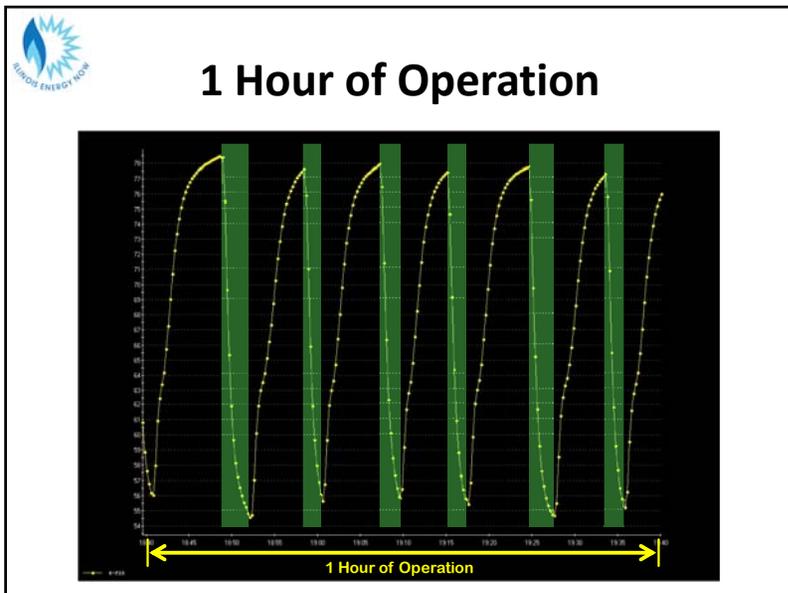
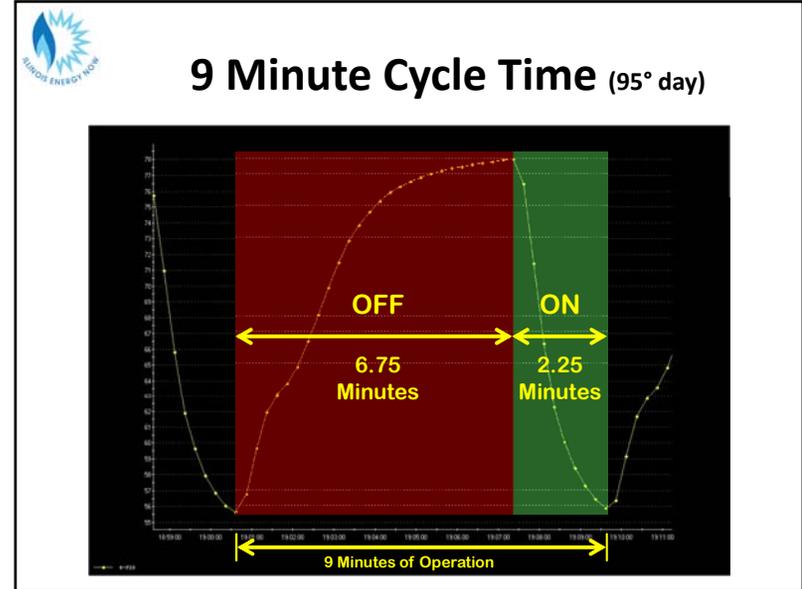
HEATING EQUIPMENT			COOLING EQUIPMENT		
Make	American Standard	Model	ADP	Make	American Standard
Trade	AMERICAN STANDARD	Model	44T83042A1	Trade	ADP
GAMA ID	2016732	Model	44T83042A1	Cond	44T83042A1
		Model	44T83042A1	Coil	HTE430,9W0**UD1D140A9H5
		Model	44T83042A1	AFR ref no.	3797072
Efficiency	92.1 AFUE	SEER	11.5	SEER	14
Heating input	60000 Btu/h	Sensible cooling	28350	Btu/h	
Heating output	55000 Btu/h	Latent cooling	12150	Btu/h	
Temperature rise	38 °F	Total cooling	40500	Btu/h	
Actual air flow	1250 cfm	Actual air flow	1350	cfm	
Air flow factor	0.028 cfm/Btu/h	Air flow factor	0.050	cfm/Btu/h	
Static pressure	0 in H2O	Static pressure	0.050	in H2O	
Space thermostat		Load sensible heat ratio	0.85		

ROOM NAME	Area (ft <sup>2</sup> )	Htg load (Btu/h)	Cig load (Btu/h)	Htg AVP (cfm)	Cig AVP (cfm)
OFFICE	120	2077	691	127	94
FOYER	74	1176	73	73	100
BR 1	132	2014	2015	139	20
BATH	68	1216	307	34	63
KITCHEN	168	3095	1656	86	330
BREAKFAST NOOK	138	8391	6823	233	200
GREAT ROOM	498	7065	4008	213	21
WIC	66	1364	421	36	96
HALLS BR	208	1216	6305	256	315
LA					
<b>GXT036C</b>	Full Load	28,600	4.4	39,300	19.8
	Part Load	22,600	4.4	29,600	20.7
Enter/leave loads		0	0	0	1960
Other equip loads		0	26431		
<b>TOTALS</b>	1651	48639	31389	1350	1350

 **2 Story Office Building**  
*(with an efficient building shell)*



**13 condensers**





Thank you!

Questions?



Darren Meyers, P.E., CEM, BPI-BA  
**International Energy Conservation Consultants**  
Education, Energy Engineering, Building Diagnostics, Sustainability  
[dmeyers@ieccode.com](mailto:dmeyers@ieccode.com)  
[www.ieccode.com](http://www.ieccode.com)